

# PARAMETRIC ANALYSIS OF LARGE DIAMETER MECHANICAL FASTENER IN REINFORCED GLULAM

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Research deals with a 49.0 mm in diameter mechanical fastener in reinforced laminated timber, to which a load perpendicular to grain is applied. Influence of glass fibre textile for reinforcing connection zone on the joint mechanical properties was carried out. The experimental tests were divided in two stages: first stage was carried out on locally reinforced and second on non-reinforcement timber specimens. Finite element models were made in software package Abaqus/CEA ver. 6.10, UMAT subroutine and surface-based cohesive behaviour were used for modelling complex timber behaviour. Results obtained by finite element model and experimental tests were shown and compared for non-reinforced and reinforced specimens with 900.0 g/m<sup>2</sup> of textile in connection zone. Also, parametric finite element model analyses were carried out for reinforced specimens with textile quantity between 0 – 900.0 g/m<sup>2</sup> in the connection zone. Research results have shown that the application of textile in connection zone increases maximum force and yield force even in the smallest quantity.

**Keywords:** large diameter fastener, mechanical fastener, reinforced timber joint, textile reinforcement, UMAT subroutine

## Parametarska analiza priključka sa spajalom velikog promjera u ojačanom lameliranom drvu

U radu su prikazana istraživanja na priključcima sa spojnim sredstvom promjera 49.0 mm. Istraživanja su provedena eksperimentalno na dvije grupe uzoraka: uzorcima s lokalnim ojačanjem drva u zoni spoja i bez lokalnog ojačanja. Uzorci su ispitani s unosom sile u spoj okomito na vlakanca drvenog elementa. Svrha provedenih istraživanja je analiza utjecaja količine tkanine na mehaničke karakteristike priključka. Obzirom da su eksperimentalna istraživanja provedena na lokalno ojačanim uzorcima sa 900.0 g/m<sup>2</sup> tkanine, područje između ojačanih i neojačanih uzoraka nadopunjeno je rezultatima iz parametarske numeričke analize. Za numeričku analizu priključaka metodom konačnih elemenata korišten je Abaqus/CEA programski paket. Za modeliranje kompleksnih modova otkazivanja, mehaničkih karakteristika i otvaranja pukotine u drvu korištena je UMAT podrutina i kohezijska interakcija između kontaktnih ploha. Rezultati istraživanja pokazali su da primjena tkanine čak i u najmanjoj količini povećava maksimalnu silu i silu popuštanja priključka.

**Ključne riječi:** lokalno ojačani drveni spoj, mehanička spajala, spajalo velikog promjera, spoj ojačan tkaninom, UMAT podrutina

## 1 Introduction

First researches on locally reinforced joints with textile in the connection zone were conducted in 80's of the last century, but since then progress in the textile application for reinforcing timber joints has not been made. Furthermore, locally reinforced joints have not been included in the European timber design code (EC5) [1]. There are few reasons for previously mentioned situation: reinforcing of joints requires additional action in the timber member production process and addition planning and controlling of locally reinforced members through each phase of the manufacturing process.

Proper application of the mechanical fasteners provides timber structures with sufficient rigidity and ductility which can be achieved by applying fasteners minimum edge and end distance and distance between two adjacent fasteners. Required distance for fasteners recommended by EC5 often requires increasing of elements dimensions at the place of joints. Consequently, that leads to increasing of dimensions of structural members and increasing of the timber volume quantity used per square meter of a structure floor area.

Previously mentioned is applicable on the mechanical fastener 20.0 mm in diameter or smaller. However, if the diameter of the fastener increases above 30.0 mm in diameter, recommended edge and end distance often exceeds the height of the member in which the fastener should be installed. Mentioned problem arises for the mechanical fastener 50.0mm in diameter, which should be

installed into the member 200.0 mm high. For mentioned type of joint, to achieve adequate structural rigidity and ductility, locally reinforced joints have necessary application.

This paper presents the research conducted on the locally reinforced joints with glass fibre textile in the connection zone. Load perpendicular to grain was applied and one mechanical fastener 49.0 mm in diameter was used. Results obtained by experimental tests were used for finite element model (FEM) calibration. Furthermore, with calibrated FEM, parametric analyses were carried out in order to define influence of textile quantity on ductility  $D$  and slip modulus  $k_s$  of joint.

This research was carried out with intention to get better insight into the influence of textile quantity on the behaviour of timber joints.

## 2 State of the art

The first researches that dealt with the reinforcement of timber structural members were carried out on timber beams with reinforced tensile zone, using steel and aluminium rods and strips [2-4]. Later studies carried out by Rowland on reinforcing beams using several types of fabrics and glues gave significant results such as increased stiffness and resistance of the beams. The research results also highlighted economic viability of such reinforced timber elements and the possibility of using fabrics as reinforcement at the connection zone [5].

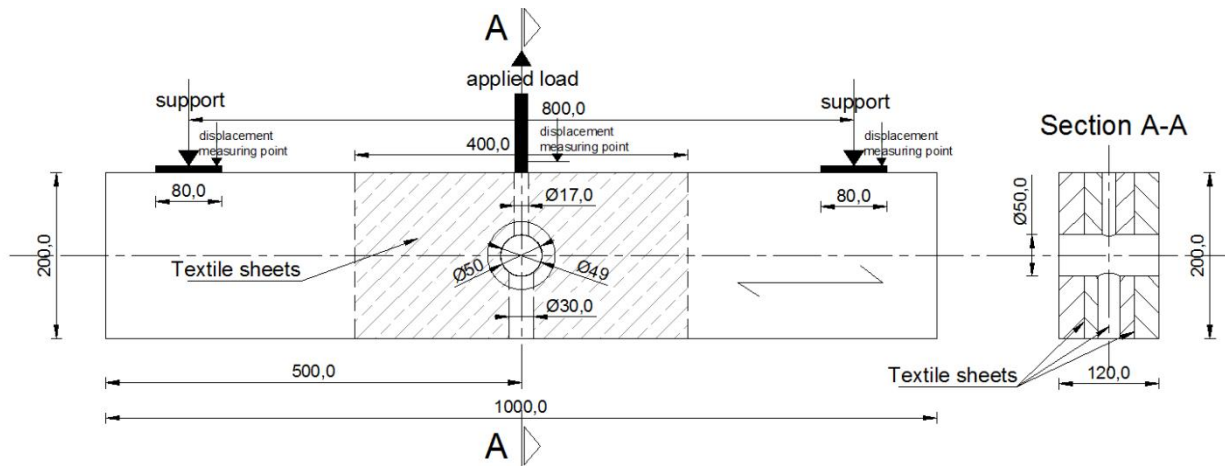


Figure 1 Schematic representation of the timber specimen with boundary conditions

Conducted research was mainly aimed at increasing ductility and reducing the edge and end distance using textile as a reinforcement of a timber element [6-8]. Screws, dowels and nails were used for fasteners in these studies.

The research carried out on locally reinforced joints using large diameter mechanical fastener was presented in papers [9, 10]. These studies were conducted using a mechanical fastener 50.0 mm in diameter while the applied load in both studies was parallel to grain. For the above-mentioned studies the textile was specially designed and timber lamellas had grooves for placing the textile.

One of the recent research with applied load parallel to grain, using a fastener 90.0 mm in diameter, and by screws reinforced timber was presented by Peter Kobel [11].

As was mentioned earlier, design equation for reinforced joints wasn't included in EC5. Also, EC5 does not contain design equations for larger diameter mechanical fasteners. K.W Johansen in 1949 [12] proposed design equations for fasteners up to 30.0 mm in diameter and since then small changes have been made on these equations which are now implemented in EC5.

Also significant researches were carried out with FEM which at the present stage of software and hardware development obtain results with satisfactory accuracy. Consequently, for timber modelling in papers [13-16] major guideline techniques are pointed out.

### 3 Experimental research



Figure 2 Specimen boundary conditions

Experimental researches were divided into two groups: specimens with locally reinforced and specimens with non-reinforced timber elements. Four specimens in each group were prepared and tested.

Four lamellas 210.0 mm wide and 32.0 mm thick were glued together with Casco adhesives MUF system 1247/2526 manufactured by Casco adhesives AB, Stockholm, Sweden. Then they were milled in order to obtain samples that were geometrically of the same size, 120.0 mm wide, 200.0 mm high, and 1000.0 mm long.

Specimens for this research were in rotated position for 90°, i.e. with vertically oriented lamellas, which is quite different from traditional laminated timber usage. The produced laminated timber was classified as GL 24h, according to EN 1194.

Regarding locally reinforced specimens, fibre glass textile sheet 200.0 mm wide and 400.0 mm long was inserted into three adhesive layers. Dimensions of those specimens with the corresponding holes are shown in Fig. 1.

Material quality used for steel parts was selected according to the stresses obtained from preliminary FEM analyses. Profiled washer, connection bush and tubes were made from steel 42CrMo4 according to EN-10027-2, with ultimate strength  $f_u=1075.0 \text{ N/mm}^2$  and yield strength  $f_y=925.0 \text{ N/mm}^2$ . For elements that were not exposed to stresses like safety rings, steel S235 JRCu with ultimate strength  $f_u=535.0 \text{ N/mm}^2$  and yield strength  $f_y=596.0 \text{ N/mm}^2$  was used. Each steel quality group was experimentally tested on specimens. Allen bolt M16 12.9 with mark DIN 912 as part of joint was used for load applying. Glass fiber textile used for reinforcing timber specimens was RT-900 K 2/2, and the fabric weight was  $900 \text{ g/m}^2 \pm 5\%$ .

Testing of the specimens was conducted on the Shimatzu AG X-300 device (Fig. 2). To achieve boundary conditions of the specimens, steel base device was mounted in the stationary lower jaw of the test machine as shown in Fig. 2. Axial spacing of the supports was 800.0 mm, and each support was 80.0 mm wide.

Tests were carried out by applying a controlled load with a constant rate 5.0 kN/min. When the load reached 25.0 kN, relaxation of 10.0 kN/min followed, until the force of 10.0 kN was reached.

Subsequent to relaxation, constant load with constant rate 5.0 kN/min was applied on the specimens to the

failure. During tests every 10.0 s load and displacement values were recorded. Bolt and supports displacements were recorded during the tests. Difference in recorded displacements, between measuring points was compared to the applied load.

During the experiment testing, average humidity value measured in the specimens was  $11,5 \% \pm 1 \%$ , and surrounding temperature was  $25^{\circ}\text{C} \pm 1^{\circ}\text{C}$ .

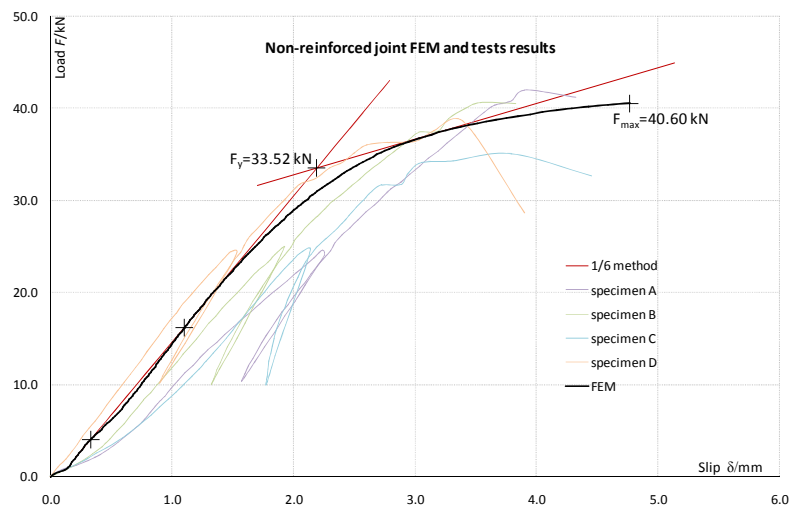
Results for each specimen, obtained by experimental tests were analyzed by 1/6 method according to EN 12512. In this method yield force  $F_y$  is determined by intersection of the following two lines: first line which is determined with point on load-slip curve corresponding to  $0.1F_{max}$  and point on load-slip curve corresponding to  $0.4F_{max}$  and second line which is tangent on load-slip curve and has an inclination of 1/6 of the first curve. Two characteristic lines by method 1/6 applied on FEM results

are shown in Fig. 3 & 4 (read lines). Furthermore, from represented results in Fig. 3 & 4, joint ductility  $D$  and slip modulus  $k_s$  were obtained as:

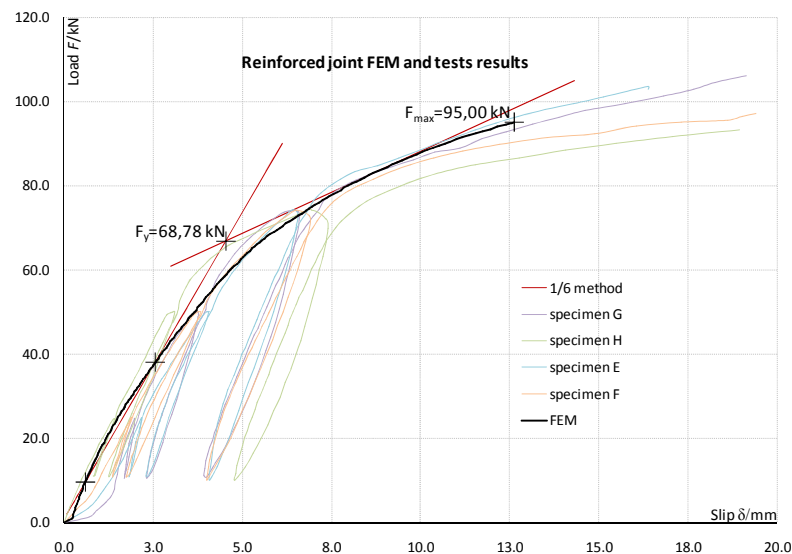
$$D = \frac{\delta_{max}}{\delta_y} \quad (1)$$

$$k_s = \frac{0.4F_{max} - 0.1F_{max}}{\delta_{0.4} - \delta_{0.1}} \quad (2)$$

where  $F_{max}$  is maximum force obtained by tests,  $\delta_{0.4}$  and  $\delta_{0.1}$  are displacements at 40% and 10% of maximum force respectively,  $\delta_{max}$  is displacement at maximum force, and  $\delta_y$  is displacement at yield force. Main values of mechanical properties obtained by tests are presented in Tab. 1 for non-reinforced specimens and in Tab. 2 for reinforced specimens.



**Figure 3** Graphical presentation of non-reinforced joint experimental and FEM results



**Figure 4** Graphical presentation of reinforced joint experimental and FEM results

**Table 1** Experimental tests results obtained on non-reinforced specimens

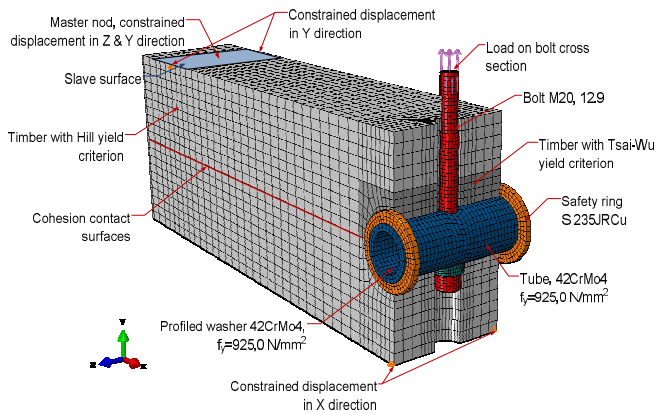
Specimens	Specimen A	Specimen B	Specimen C	Specimen D
Slip modulus $k_s/\text{kN/mm}$	13.26	15.24	12.23	17.58
Ductility $D/-$	1.19	1.33	1.24	1.72
Max. force $F_u/\text{kN}$	42.0	40.65	35.07	38.97

**Table 2** Experimental tests results obtained on reinforced specimens

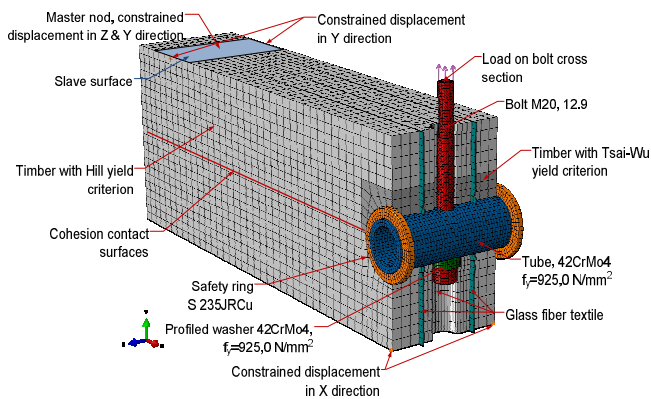
Specimens	Specimen E	Specimen F	Specimen G	Specimen H
Slip modulus $k_s$ /kN/mm	15.73	15.53	20.39	16.34
Ductility $D/-$	3.13	4.21	4.55	5.11
Max. force $F_u$ /kN	103.65	97.26	106.11	93.28

#### 4 Finite element models

For both groups of specimens FEM were made in the software package ABAQUS/CAE ver. 6.10. Numerical models were created with first-order volume finite elements C3D8 (Fig. 5. & 6.). These finite elements are defined with eight nodes and they are using full integration method. For defining non-reinforced timber specimen geometry 25 096 FE was used, and for reinforced timber specimen 39 602 FE. Steel parts in both numerical models were defined with 32 558 FE.

**Figure 5** Non-reinforced joint FEM

Boundary conditions for both specimens were defined as presented in Fig. 5 & 6. For supports modeling, master-slave function was used with constrained displacement in Z and Y direction of master nodes. Slave surfaces were defined with mutual center distance of 800.0 mm, and each surface was 80.0 mm long and 120.0 mm wide. To ensure specimen stabilization five more nodes were constrained in Y and X direction.

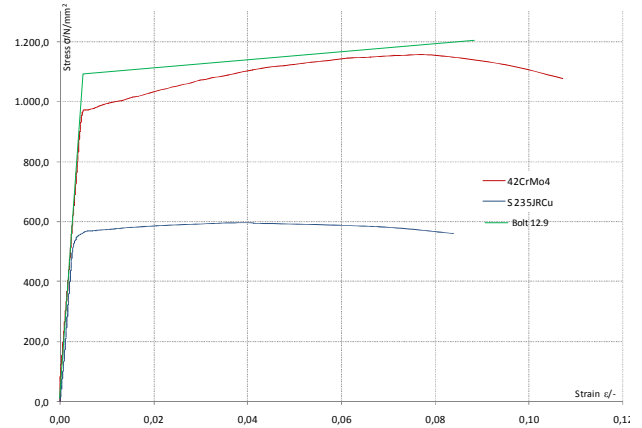
**Figure 6** Reinforced joint FEM

Analyses were conducted with Full Newton solution technique, and automatic control of load step increment, while maximum load increment was set to 0.1 N/t. Load was applied with stress of 6.49612 N/mm<sup>2</sup> on Allan bolt cross section. Applied stress corresponds on the bolt cross section area of 153.0 mm<sup>2</sup> to 1.0 kN.

For contact modelling between two joint parts, absolute rigidity in the normal direction was used and for tangential direction coefficient of friction 0.2 for steel-steel and 0.25 steel-timber contact was used. [15].

The textile fabric inserted into the adhesive layer for experimental research was modelled using C3D8 volume FE. In order to achieve the accuracy of the FEM, the ratio between the sides of FE was maintained 1:8, using the fabric 1.0 mm thick. For textile sheets, bilinear mechanical behavior was used (Fig. 7).

Textile specimens were tested, for load in warp and weft direction, and average failure loads were obtained  $P_{ul}$ =154.78 N/mm for warp direction and  $P_{ul}$ =161.38 N/mm for weft direction. Failure loads obtained by tests were recalculated to textile ultimate strength for thickness of 1.0 mm. Modulus of elasticity (MOE)  $E$ =4 571 N/mm<sup>2</sup>, share modulus  $G$ = 650.0 N/mm<sup>2</sup>, Poisson ratio  $\nu$ =0.38 and ultimate strength  $f_u$ =160.0 N/mm<sup>2</sup> were used in FEM for textile sheet.

**Figure 7** Steel stress-strain diagram

Steel parts in FEM were modeled with elasto-plastic mechanical properties. Mechanical properties  $E$ =210000 N/mm<sup>2</sup> and  $\nu$ =0.3 were used for all steel parts. Steel strengths is pointed out in Chapter 3 and stress-strain diagram are shown in Fig 7. For all steel parts except bolt, plastic region was defined thru 17 points, which are corresponding to average stress-strain values obtained from experimental tests. Bolt was modeled as bi-linear with tangential modulus  $E^t=E \times 0.05$ . Steel parts were modeled both in FEM with C3D8 FE defined with eight nodes.

For defining yield criteria and orthotropic elasto-plastic wood behaviour, with different mechanical properties in tension and compression, UMAT subroutine was defined and used. Modified Hill yield criterion [17] was used as the basic wood yield criterion in UMAT subroutine and for timber around fastener Tsai-Wu yield criterion was used.



Tsai-Wu yield criterion can be expressed as:

$$\begin{cases} F_1\sigma_L + F_2\sigma_R + F_3\sigma_T + F_{11}\sigma_L^2 + F_{22}\sigma_R^2 + F_{33}\sigma_T^2 \\ + F_{44}\tau_{LR}^2 + F_{55}\tau_{LT}^2 + F_{66}\tau_{RT}^2 = 1.0/0.7/0.0 \\ F_1 = \frac{1}{f_{u,L}^t - f_{u,L}^c}; F_2 = \frac{1}{f_{u,R}^t - f_{u,R}^c}; \\ F_3 = \frac{1}{f_{u,T}^t - f_{u,T}^c}; F_{11} = \frac{1}{f_{u,L}^t \cdot f_{u,L}^c}; \\ F_{22} = \frac{1}{f_{u,R}^t \cdot f_{u,R}^c}; F_{33} = \frac{1}{f_{u,T}^t \cdot f_{u,T}^c}; \\ F_{44} = F_{55} = F_{66} = \frac{1}{f_{u,S}^2} \end{cases} \quad (3)$$

where  $\sigma_L, \sigma_R, \sigma_T$  are normal stresses in longitudinal, radial and transversal directions respectively,  $\tau_{RT}, \tau_{TL}, \tau_{LR}$  are shear stresses in corresponding planes,  $f_{u,L}^t, f_{u,R}^t, f_{u,T}^t$  are wood tensile strengths respectively in longitudinal, radial and transversal directions,  $f_{u,S}$  is shear strength, and  $f_{u,L}^c, f_{u,R}^c, f_{u,T}^c$  are wood compression strength, respectively in longitudinal, radial and transversal directions.

$$\begin{cases} A \cdot (\sigma_R - \sigma_T)^2 + B \cdot (\sigma_T - \sigma_L)^2 + C \cdot (\sigma_L - \sigma_R)^2 + \\ 2D \cdot \tau_{RT}^2 + 2E \cdot \tau_{TL}^2 + 2F \cdot \tau_{LR}^2 \leq 1 \\ A = \frac{1}{2} \left[ \frac{1}{(f_{u,R}^t)^2} + \frac{1}{(f_{u,T}^t)^2} - \frac{1}{(f_{u,L}^t)^2} \right], \\ B = \frac{1}{2} \left[ \frac{1}{(f_{u,T}^t)^2} + \frac{1}{(f_{u,L}^t)^2} - \frac{1}{(f_{u,R}^t)^2} \right], \\ C = \frac{1}{2} \left[ \frac{1}{(f_{u,L}^t)^2} + \frac{1}{(f_{u,R}^t)^2} - \frac{1}{(f_{u,T}^t)^2} \right], \\ D = \frac{1}{2 \cdot (f_{u,RT}^t)^2}, E = \frac{1}{2 \cdot (f_{u,TL}^t)^2}, F = \frac{1}{2 \cdot (f_{u,LR}^t)^2}. \end{cases} \quad (4)$$

Hill yield criterion [17] expressed by Eq. (4) is modified for compression and for tension separately, because from Eq. (4) arises the following condition: if the stress in one direction reaches yield point, then for the other two directions yield point can be assumed. Much more accurate assumption for tension would be if Eq. (4) is defined for each direction separately. Now, yield criteria for tension are obtained [13, 15] as follows:

$$\begin{cases} \frac{\sigma_L^2}{f_{u,L}^t{}^2} + \frac{\tau_{LT}^2 + \tau_{LR}^2}{f_{u,S}^2} = 1, & \frac{\sigma_R^2}{f_{u,R}^t{}^2} + \frac{\tau_{LR}^2 + \tau_{RT}^2}{f_{u,S}^2} = 1, \\ \frac{\sigma_T^2}{f_{u,T}^t{}^2} + \frac{\tau_{LT}^2 + \tau_{RT}^2}{f_{u,S}^2} = 1 \end{cases} \quad (5)$$

Yield criterion set by Eq. (5) cannot be applied for compression stresses, because if the yield occurs in wood under compressive stress in one direction, then the yield assumption must be applied on the other two directions.

A yield criterion of maximum principal stress is chosen for wood under compression as follows:

$$\frac{\sigma_L^2}{f_{u,L}^c{}^2} = 1, \quad \frac{\sigma_R^2}{f_{u,R}^c{}^2} = 1, \quad \frac{\sigma_T^2}{f_{u,T}^c{}^2} = 1 \quad (6)$$

It is important to notice a resemblance between Hill and Hashin yield criterion (1980), which is proposed for materials similar to wood [18]. Eq. (5,6) are implemented in UMAT subroutine for wood yield criterion and they are in detail explained in [19, 20]. Wood elastic mechanical properties and mail ultimate strengths used in FEM and pointed out in Tab 3, are emanated from laboratory tests which are presented in [19, 21, 22] and literature.

Also, for defining mechanical properties of textile sheets, UMAT subroutine was used. Because glass fiber textile in compression has no strength, it can be assumed that mechanical properties of wood with Tsai-Wu criterion can be applied on textile under compression. For textile in tension, maximum principal stress criterion was used.

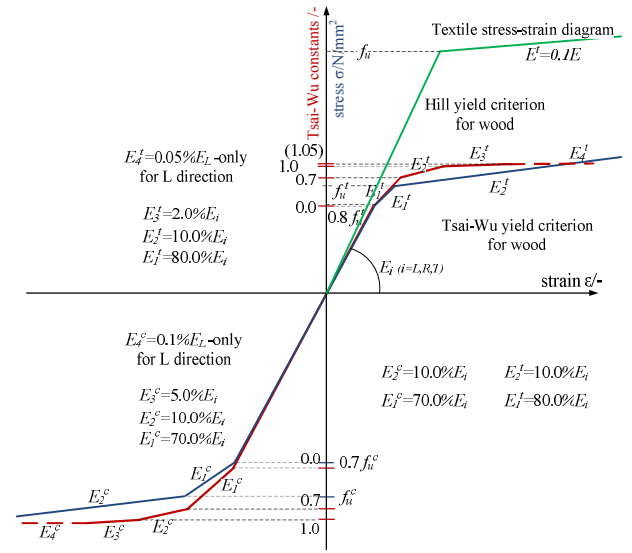


Figure 8 Stress-strain diagrams for wood

Along with UMAT subroutine surface-based cohesive behavior was used for defining crack opening in wood. Surface-based cohesive behavior is very similar to the cohesive elements with traction-separation response [23] and is defined by interfacial penalty stiffness  $E_i^0$  ( $i=1,2,3$ , normal, the first, and the second shear direction, respectively), interfacial strength  $\sigma_i^0$ , and critical fracture energy  $G_i^c$ . Fig. 9 shows traction separation law, which consists of elastic deformation, damage initiation criterion and linear damage evaluation law. Prior to the initiation of damage, elastic deformation will be produced and can be expressed as:

$$\delta_i^o = \frac{\sigma_i^o}{E^o}; \quad i=1,2,3 \quad (7)$$

Interfacial penalty stiffness is taken as  $E^0=20\,000$  N/mm and it is not material-related parameter. Damage initiation starts when the contact stresses satisfies a specified damage initiation criterion. In this case quadratic nominal stress criterion is used for damage initiation, and it is defined by Eq. (8).

**Table 3** Wood material properties and main ultimate strengths

Elastic modulus/MPa	Shear modulus/MPa	Poisson's ratio	Compression strength /MPa	Tension strength /MPa	Share strength/MPa
$E_L=11500.0$	$G_{LR}=600.0$	$\nu_{LR}=0.42$	$f_{uL}^c=26.4$	$f_{uL}^t=19.8$	
$E_T=450.0$	$G_{RT}=60.0$	$\nu_{RT}=0.50$	$f_{uT}^c=2.97$	$f_{uT}^t=0.38$	$f_{uS}=3.3$
$E_R=600.0$	$G_{LT}=650.0$	$\nu_{LT}=0.48$	$f_{uR}^c=2.97$	$f_{uR}^t=0.38$	

$$1 = \left\{ \frac{\langle \sigma_1 \rangle}{\sigma_1^0} \right\}^2 + \left\{ \frac{\sigma_2}{\sigma_2^0} \right\}^2 + \left\{ \frac{\sigma_3}{\sigma_3^0} \right\}^2 \quad (8)$$

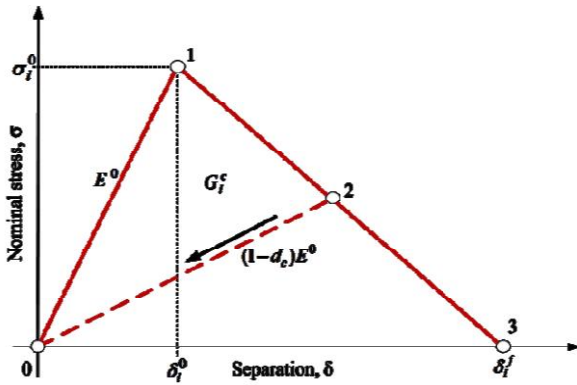
where  $\langle \sigma_1 \rangle$  indicates that a pure compression stress state will not initiate damage. Total deformation for cohesion elements until crack appears can be expressed as follows:

$$\delta_m = \sqrt{\langle \delta_1^2 \rangle + \delta_2^2 + \delta_3^2} \quad (9)$$

Assuming that two shear directions have equal mechanical properties, expression for the crack deformation can be expressed as follows [24, 25]:

$$\delta_2^0 = \delta_3^0 = \delta_{pos}^0; \quad \beta = \frac{\delta_{pos}}{\delta_1}$$

$$\delta_m^0 = \begin{cases} \delta_1^0 \delta_2^0 \sqrt{\frac{1+\beta^2}{(\delta_2^0)^2 + (\beta \cdot \delta_1^0)^2}} \leftarrow \delta_1^0 > 0 \\ \delta_{pos}^0 \leftarrow \delta_1^0 \leq 0 \end{cases} \quad (10)$$



**Figure 9** Bilinear constitutive model for crack opening

Using corresponding energy release values shown in Fig. 9, total deformation at complete failure can be expressed for specific directions as follows:

$$\delta_i^f = \frac{2 \cdot G_i^c}{\sigma_i^0}; \quad i=1,2,3 \quad (11)$$

Mixed-mode fracture energy dependence can be defined by a power law fracture criterion. Power law criterion states that failure under mixed-mode condition is governed by a interaction of the energies required to cause failure in the individual modes.

Criterion can be written as follows:

$$\left\{ \frac{G_1}{G_1^c} \right\}^\alpha + \left\{ \frac{G_2}{G_2^c} \right\}^\alpha + \left\{ \frac{G_3}{G_3^c} \right\}^\alpha = 1 \quad (12)$$

In Eq. (12) quantities  $G_1$ ,  $G_2$ ,  $G_3$  refer to work done by the traction and conjugate relative displacement in normal and two share directions. Quantities  $G_1^c$ ,  $G_2^c$ ,  $G_3^c$  need to be specified, and they refer to critical energies required to cause failure in certain direction.

From the above mentioned the following equation for the total deformation at complete failure can be expressed by the dissipated energy criterion:

$$\delta_m^f = \begin{cases} \frac{2(1+\beta^2)}{E^c \delta_m^0} \cdot \left[ \left\{ \frac{1}{G_1^c} \right\}^\alpha + \left\{ \frac{\beta^2}{G_2^c} \right\}^\alpha \right]^{-\frac{1}{\alpha}} \leftarrow \delta_1^0 > 0 \\ \delta_{pos}^k \leftarrow \delta_1^0 \leq 0 \end{cases} \quad (13)$$

Constitutive equations for the mixed-mode failure are defined with interfacial penalty stiffness  $E$ , crack opening function  $d$  and total crack deformation  $\delta_m^f$ ,  $\delta_m^o$ , and can be expressed as follows:

$$\sigma = D_{sr} \delta_r \quad (14)$$

$$D_{sr} = \begin{cases} \overline{\delta_{sr}} E^0 & \leftarrow \delta_m^{\max} \leq \delta_m^o \\ \overline{\delta_{sr}} \left[ (1-d) E^0 + E^0 d \overline{\delta_{s1}} \frac{\langle -\delta_1 \rangle}{-\delta_1} \right] & \leftarrow \delta_m^o \leq \delta_m^{\max} \leq \delta_m^k \\ E^0 \overline{\delta_{s1}} \frac{\langle -\delta_1 \rangle}{-\delta_1} & \leftarrow \delta_m^{\max} \leq \delta_m^k \end{cases} \quad (15)$$

$$d = \frac{\delta_m^f (\delta_m^{\max} - \delta_m^o)}{\delta_m^{\max} (\delta_m^f - \delta_m^o)}, \quad d \in [0,1] \quad (16)$$

Mechanical properties for cohesion contact surfaces are taken from literature [26] and they are presented in table Tab. 4.

## 5 Results obtained by FEM

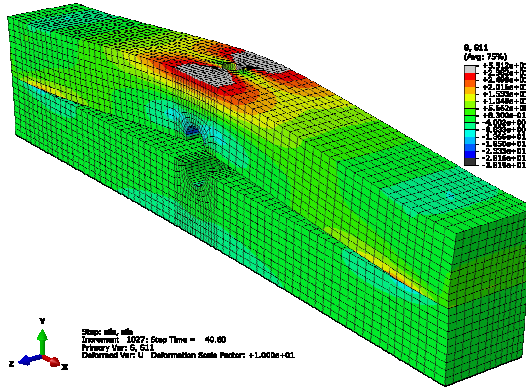
FEM results of reinforced joint with 900,0 g/m<sup>2</sup> of glass fibre textile in connection zone and non-reinforced joint, were analyzed by 1/6 method according to EN12512 norm. Obtained FEM results are presented together with experimental tests results in Fig. 3 and Fig. 4. Presented results show good overlapping in load-slip curve and small variations between tests and FEM results.

Good overlapping between results is important if we bear in mind that the same FEM were used for the parametric analyses.

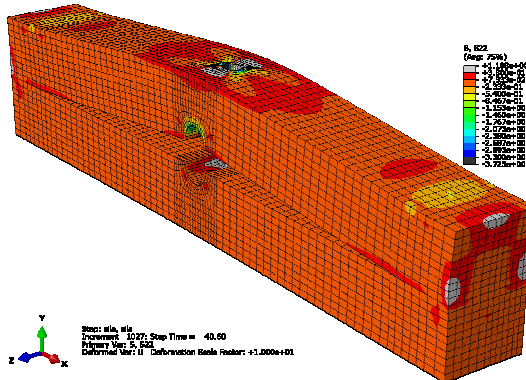
The biggest difference between FEM results and experimental tests results are noticed in the strains above yield force of the reinforced joint. In experimental tests after embedding compression strength in timber specimens was reached, strains propagation increased, and fastener sank into the timber. With Tsai-Wu yield criterion previous mentioned mode failure has been successfully obtained with FEM but with small variation. The reason for this difference is inability to accurately numerically define descending branch in the wood stress-strain diagram in compression.

**Table 4** Surface-based cohesive behavior mechanical properties

	Timber	Textile	Adhesive
$G_1^C$	0.179	1.0	4.0
$G_2^C$	0.4	3.0	6.0
$G_3^C$	0.4	3.0	6.0
$E^C$	1000.0	4500.0	10000.0
$\alpha$	1.0	1.0	1.0



**Figure 10** Stresses in direction S11 - non-reinforced specimen



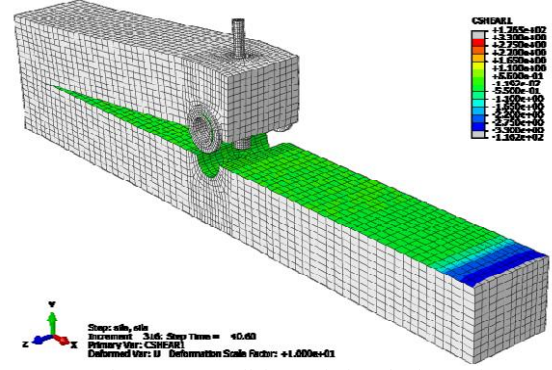
**Figure 11** Stresses in direction S33 - non-reinforced specimen

Stresses parallel (S11) and perpendicular to grain (S33) at the time of joint failure are presented in Fig. 10, 11, 14 and 15. At the time of complete joint failure, stresses parallel to grain were between the compression and tension yield stresses.

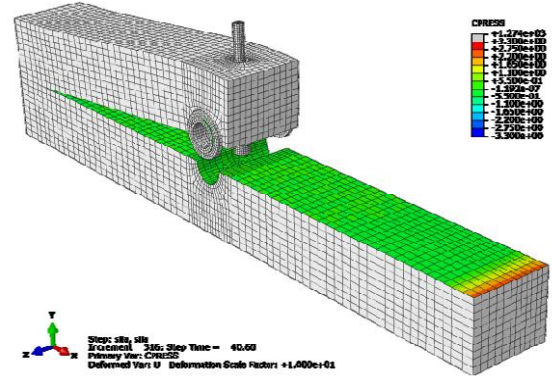
Exceeding of embedding compression strength is visible in reinforced specimen shown in Fig. 15. (stresses perpendicular to grain).

Tensile stress perpendicular to grain caused initial cracks in the middle of the timber specimen. Complete failure was caused by interaction of tensile and shear stresses parallel to grain. The stresses in the cohesion

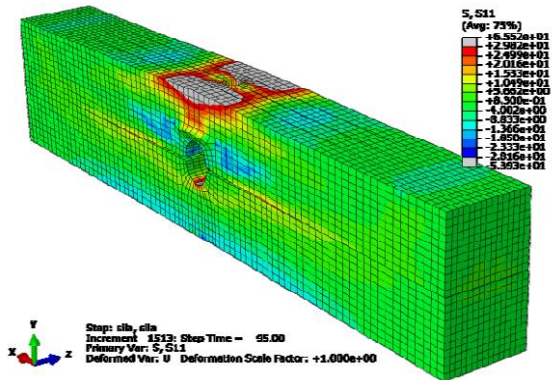
zone in the time of specimen failure are shown in Fig. 12 and Fig. 13.



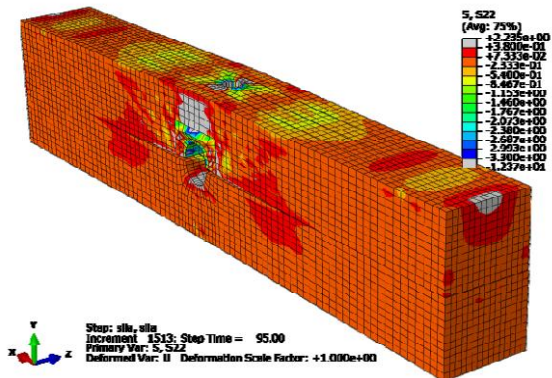
**Figure 12** Shear stress parallel to grain in cohesion surface, non-reinforced specimen



**Figure 13** Normal stress perpendicular to grain in cohesion surface - non-reinforced specimen



**Figure 14** Stresses in direction S11 - reinforced specimen with 900.0kg/m² of textile



**Figure 15** Stresses in direction S33 - reinforced specimen with 900.0kg/m² of textile



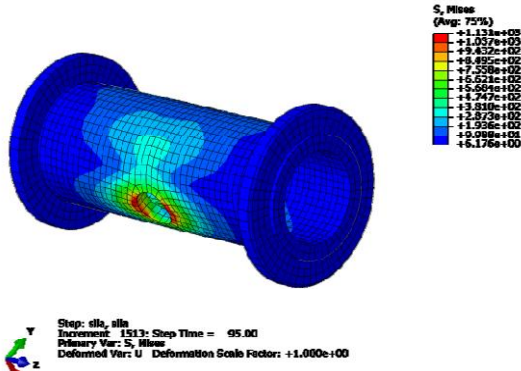


Figure 16 Von Mises stress in steel pipe at specimen failure, reinforced specimen with 900.0 kg/m<sup>2</sup> of textile

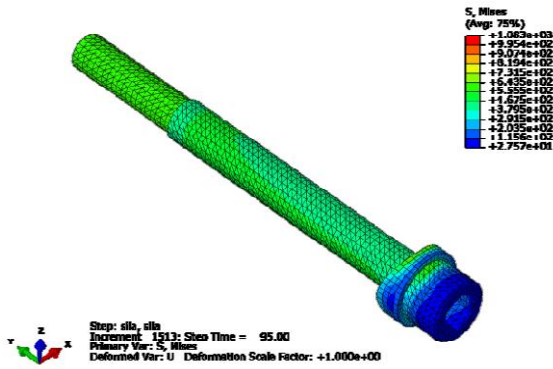


Figure 17 Von Mises stress in bolt at specimen failure, reinforced specimen with 900.0 kg/m<sup>2</sup> of textile

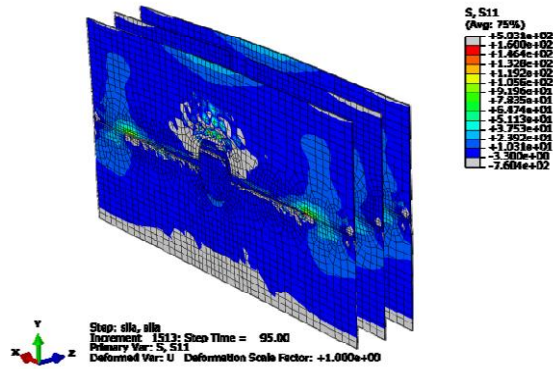


Figure 18 Stresses in textile in direction S11 (warp direction) reinforced specimen with 900.0 kg/m<sup>2</sup> of textile

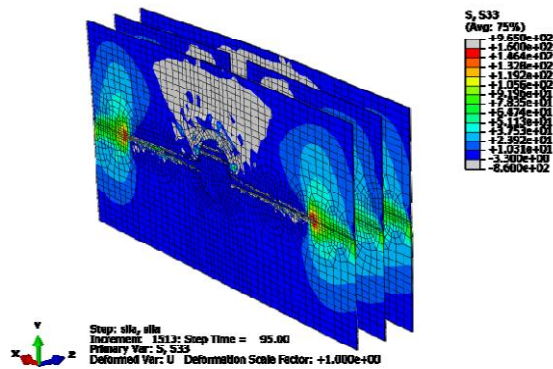


Figure 19 Stresses in textile in direction S22 (weft direction) reinforced specimen with 900.0 kg/m<sup>2</sup> of textile

The main purpose of FEM research was to calibrate material mechanical properties and mode failures. Calibrated FEM's were used for making parametric FEM analyses.

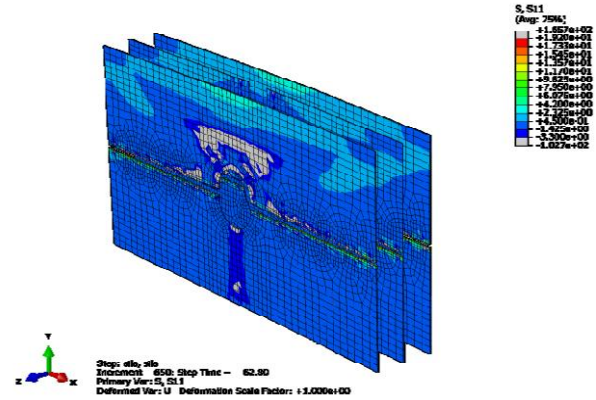


Figure 20 Stresses in textile in direction S11 (warp direction) reinforced specimen with 108.0 kg/m<sup>2</sup> of textile

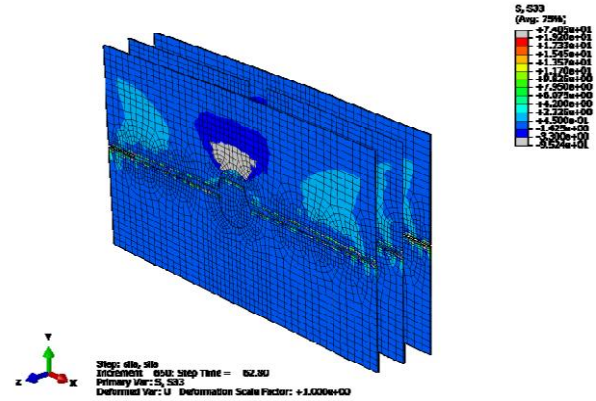


Figure 21 Stresses in textile in direction S22 (weft direction) reinforced specimen with 108.0 kg/m<sup>2</sup> of textile

Parametric FEM analyses were carried out on the reinforced joint with the following quantities for the textile in place of connection: 450.0 g/m<sup>2</sup>, 225.0 g/m<sup>2</sup> and 108.0 g/m<sup>2</sup>. Results of numerical analyses are presented in slip-load graphs for each model separately (Figs. 22 ÷ 26). All results obtained by parametric FEM (Tab. 5) were analysed by 1/6 method according to EN12512 norm.

In Tab. 5 given values were calculated using the following equations:

$$\mu = \frac{\sum m}{A \cdot t} - \text{degree of reinforcement (DOR)}, \frac{\text{kg}}{\text{m}^2 \cdot \text{m}}$$

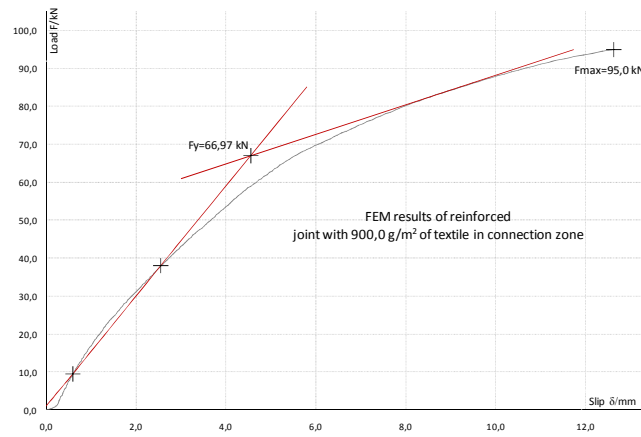
where  $\sum m$  is total textile mass in joint,  $A$  is timber reinforced surface in specimen and  $t$  is timber specimen thickness.

$$f_h = \frac{F_u}{A_f} - \text{ultimate strength}$$

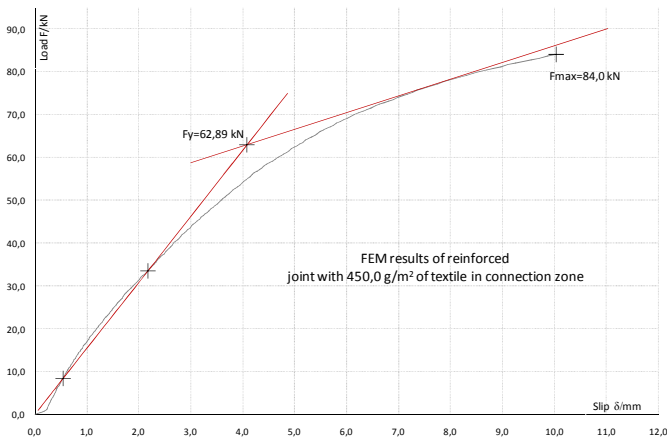
where  $A_f$  is fastener contact surface.

**Table 5** Parametric FEM results

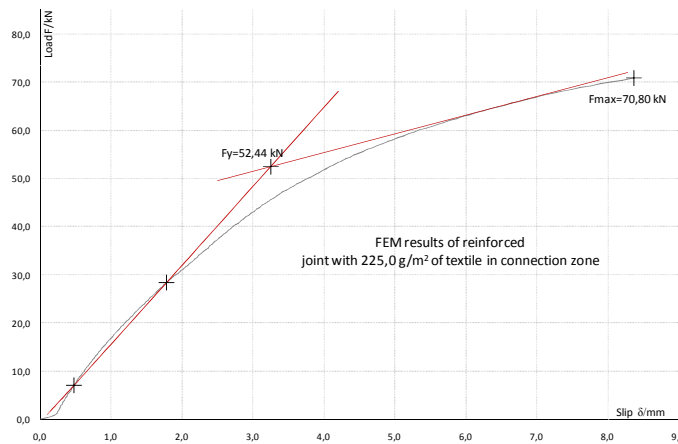
Textile quantity $m / g/m^2$	DOR $\mu / kg/m^2m$	Maximal force $F_u / kN$	Strength $f_t / N/mm^2$	Yield force $F_y / kN$	Stiffness $K_s / kN/mm$
900.0	22.50	95.0	16.711	66.97	14.47
450.0	11.25	84.0	14.78	62.89	15.46
225.0	5.625	70.8	14.40	52.44	16.34
108.0	2.70	62.8	12.62	46.28	16.82
0.0	0.0	40.6	7.47	33.49	15.82



**Figure 22** FEM results of reinforced joint with 900.0g/m<sup>2</sup> of textile in connection zone

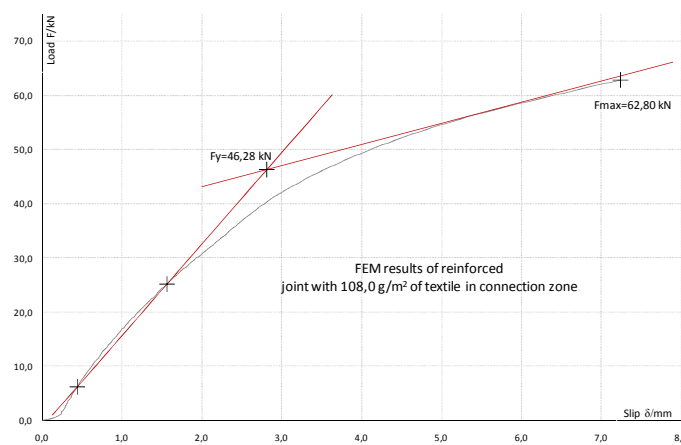


**Figure 23** FEM results of reinforced joint with 450.0g/m<sup>2</sup> of textile in connection zone

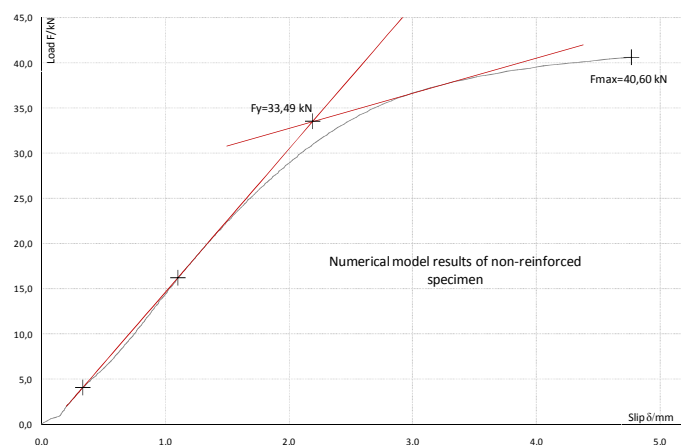


**Figure 24** FEM results of reinforced joint with 225.0g/m<sup>2</sup> of textile in connection zone

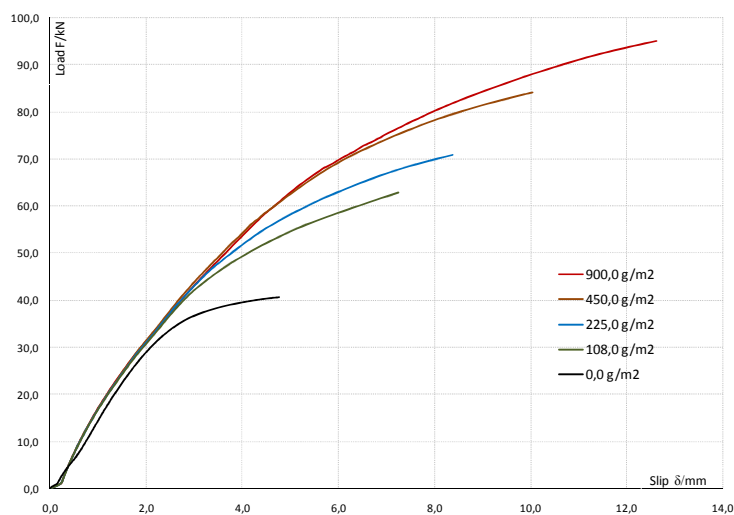




**Figure 255** FEM results of reinforced joint with 108.0g/m<sup>2</sup> of textile in connection zone



**Figure 26** FEM results of non-reinforced joint



**Figure 27** FEM results obtained by parametric analyses

## 6 Conclusion

Research results have shown that glass fibre textile placed between lamellas in adhesive layer effectively increases the resistance of the timber element as well as the tensile strength of laminated timber perpendicular to grain. In locally reinforced joints maximum force and yield force have doubled and the ductility has increased 3.24 times comparing to non-reinforced joints, which

economically justifies this way of reinforcing timber structures.

FEM with complex mechanical characteristics of timber, crack opening and propagation, presented in this paper, showed good correlation with experimental results. Such complex numerical models provide realistic behaviour of joints and were used to determine the influence of textile quantity on the joint resistance.

FEM results have shown that the resistance of the joint does not increase linearly with an increasing of

textile quantity, and thus the economic profitability of reinforced joint is limited.

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